

# Acoustics

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# The Acoustics Application



- The acoustics application mode provides two analysis types.
- Time-harmonic analysis

$$\nabla \cdot \left( -\frac{1}{\rho_0} \nabla p + q \right) - \frac{\omega^2 p}{\rho_0 c_s^2} = 0$$

Coefficient	Description
$\omega = 2\pi f$	Angular frequency
$\rho_0$	Fluid density
$c_s$	Speed of sound
$q$	Dipole source

- Eigenvalue analysis

$$\nabla \cdot \left( -\frac{1}{\rho_0} \nabla p \right) - \frac{\lambda p}{\rho_0 c^2} = 0$$

Coefficient	Description
$\lambda$	Eigenvalue



# Subdomain quantities

Quantity	Variable	description
$\rho_0$	rho0	Density
$c_s$	cs	Speed of sound
$q$	qx, qy, qz	Dipole source

- Density : density is that of the fluid in which the acoustic waves propagate. The default value is 1.024 kg/m<sup>3</sup>.
- Speed of sound : The default value is 343m/s, the speed of sound in air at approximately 20 °C.
- Dipole source : Vector with individual components for each space dimension.

# Boundary conditions

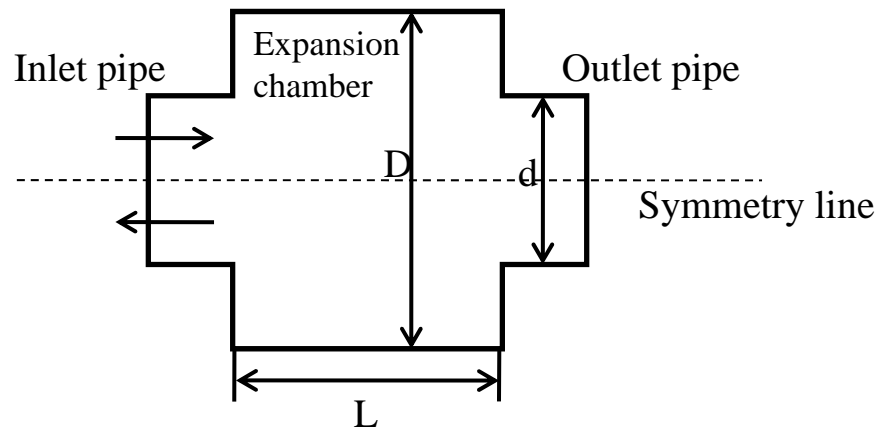


- Boundary conditions applied to the different analysis types.
  - Sound hard boundaries (walls)
  - Sound soft boundaries
  - Pressure source
  - Impedance boundary condition
  - Radiation boundary condition
  - Specified normal acceleration
  - Axial symmetry
  - Interface conditions on borders



# Model : Reactive muffler

- This model examines the sound transmission properties of an idealized reactive muffler with long inlet and outlet pipes.



- You need to include only half of the geometry as indicated in the above figure, because this is an axisymmetric model.

# Equations for reactive muffler

- Transmission loss coefficient

$$D_{TL} = 10 \cdot \log\left(\frac{W_i}{W_t}\right)$$

- Transmission loss coefficient (1D theoretical solution)

$$D_{TL} = 10 \cdot \log\left(1 + \left(\frac{S_1}{2 \cdot S_2} - \frac{S_2}{2 \cdot S_1}\right)^2 \cdot (\sin(k \cdot L))^2\right)$$

- Intensity

$$I = \frac{p^2}{2 \cdot \rho_0 \cdot c}$$

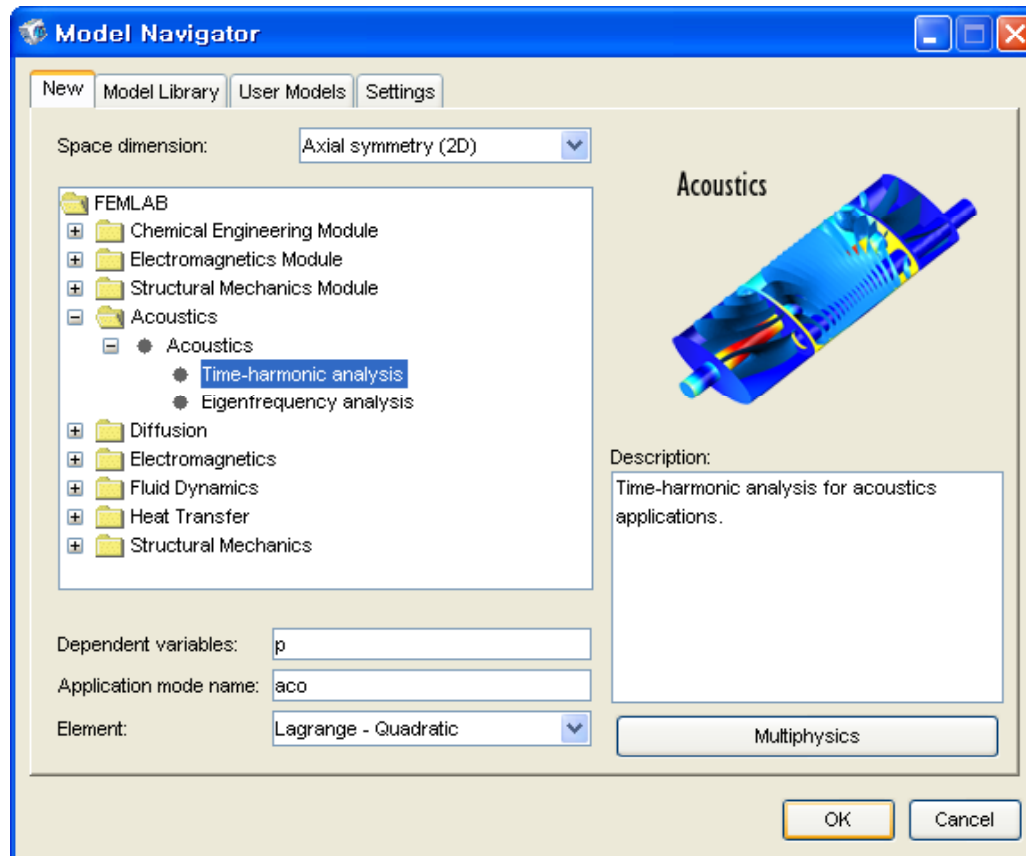
- Sound power

$$W = \int (I \cdot 2 \cdot \pi \cdot r) dr$$

- Frequency

$$f = 1.841 \cdot \frac{c}{\pi \cdot D}$$

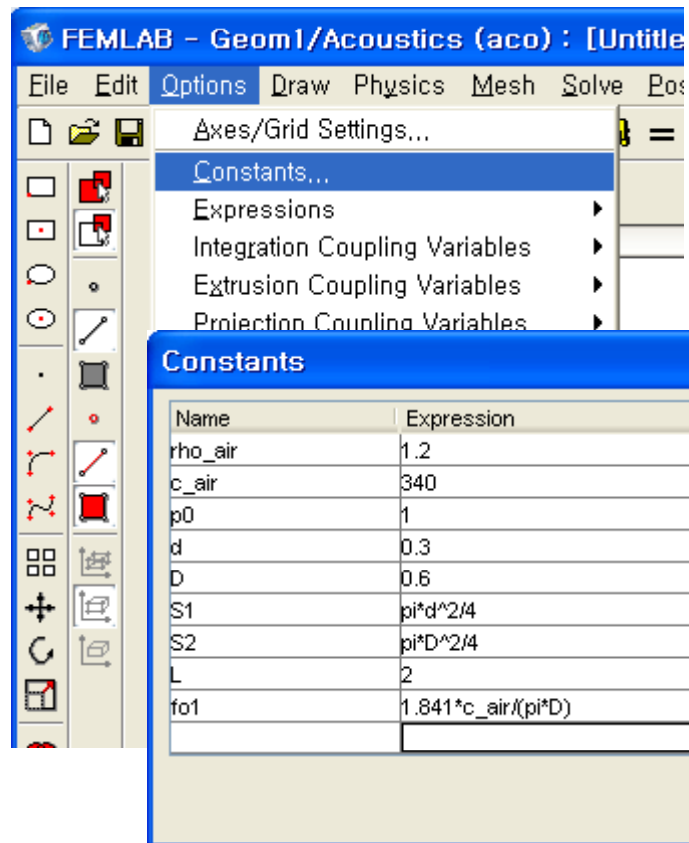
# Modeling using the FEMLAB Model navigator



1. Go to the **Model Navigator** and select **Axial symmetry (2D)** in the **Space dimension** list
2. In the list of application modes open the **Acoustics** folder and then **Time-harmonic analysis**.
3. Click **OK**.

# Modeling using the FEMLAB

## Options and settings

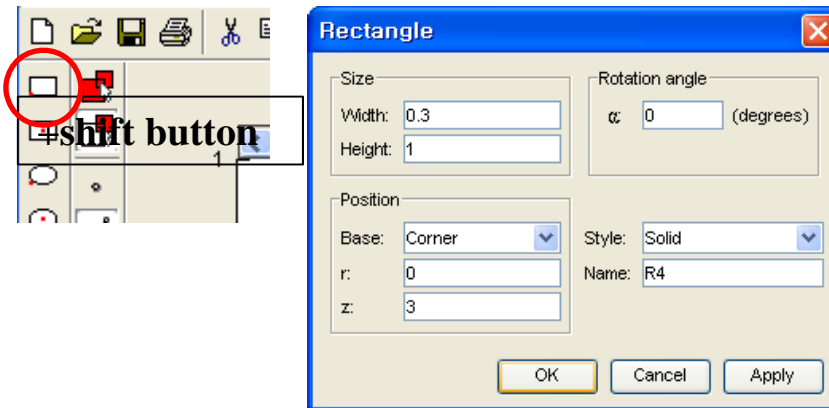


1. Go to the **Options** menu and choose **Constans** to parameterize the model.
2. In the **Constants** dialog box enter the following constants.

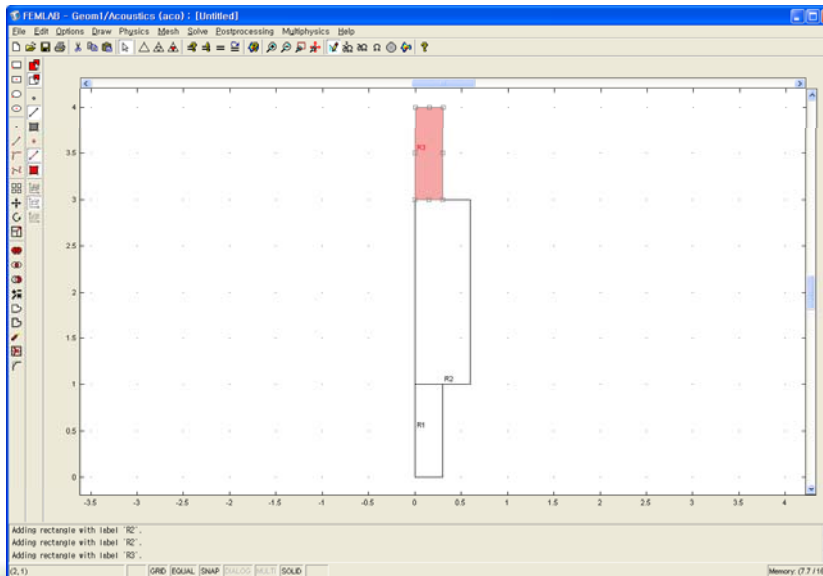


# Modeling using the FEMLAB

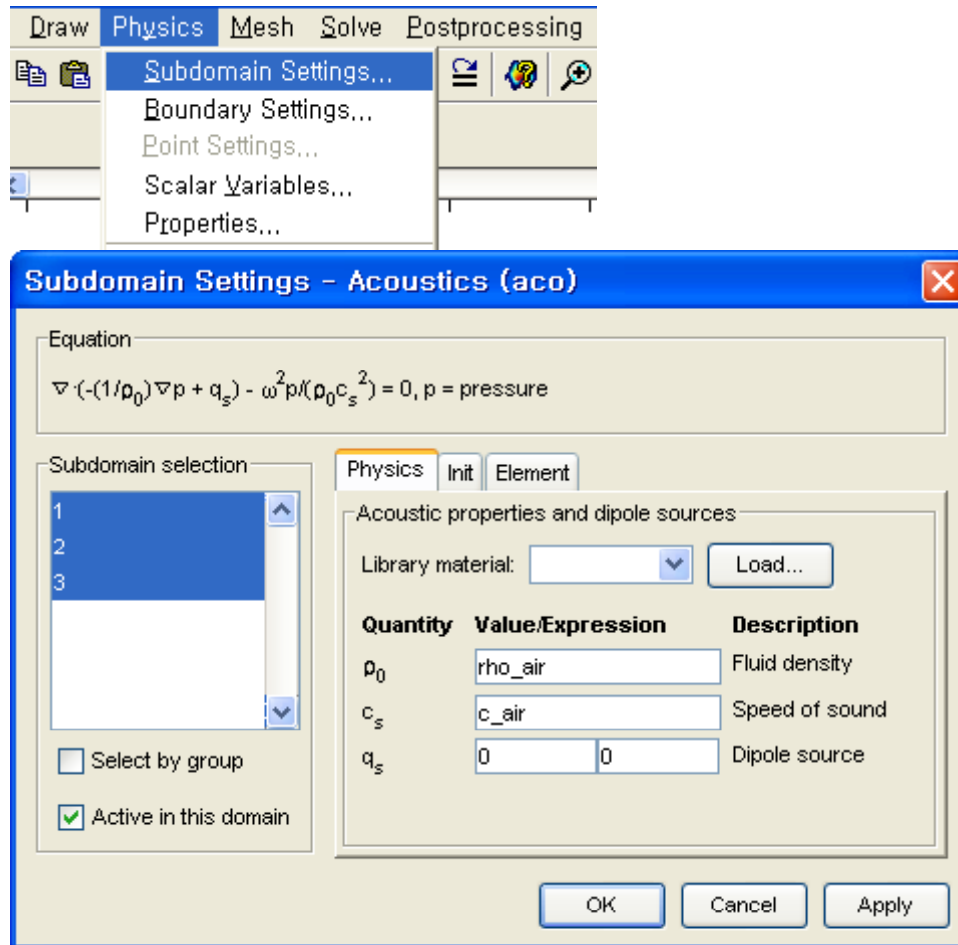
## Geometry modeling



1. Shift-click the **Rectangle/Square** button to specify a rectangle.
2. Enter 0.3, 0.6, 0.3 in the **Width** edit field and 1, 2, 1 in the **height** field and 0, 1, 3 in the **z** edit field respectively.



# Modeling using the FEMLAB Physics settings (subdomain)



1. Go to the **Subdomain** menu and choose **Subdomain Settings**.
2. Select all subdomains from the **Subdomain selection** list.
3. Enter rho\_air in the **Fluid Density** edit field.
4. Enter c\_air in the **Speed of sound** edit field.
5. Leave the default settings (0) for the **Dipole source**.

# Modeling using the FEMLAB

## Physics settings (boundary)

Boundary Settings - Acoustics (aco)

Equation  
 $n \cdot 8711p = 0$

Boundary selection

5  
6  
7  
8  
9  
10  
11  
12

Select by group  
 Interior boundaries

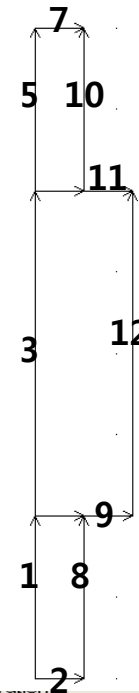
Boundary sources and constraints

Boundary condition: Sound hard boundary (wall)

Wave type: Plane wave

Quantity	Value/Expression	Description
$p_0$	0	Pressure sour
$a_n$	0	Normal accele...
Z	1.25*343	Input impedance
$r_0$	0	Source location, r coordinate
$z_0$	0	Source location, z coordinate
$\vec{k}$	-nr -nz	Wave direction

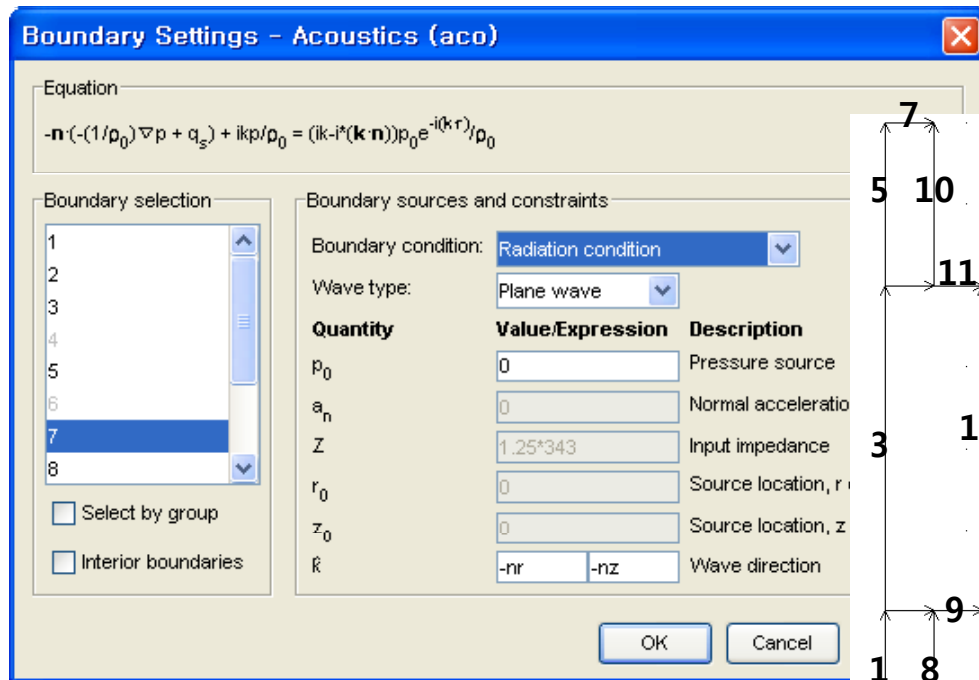
OK Cancel Apply



1. Go to the **Boundary** menu and choose **Boundary Settings**.
2. In the **Boundary Conditions** dialog box enter the following boundary coefficients
3. Select boundaries 1, 3 and 5 in the **Boundary selection** list.
4. Select Axial symmetry in the **Boundary condition** list.
5. Select boundaries 8 to 12 in the **Boundary selection** list.
6. Select Sound hard boundary (wall) in the **Boundary condition** list.

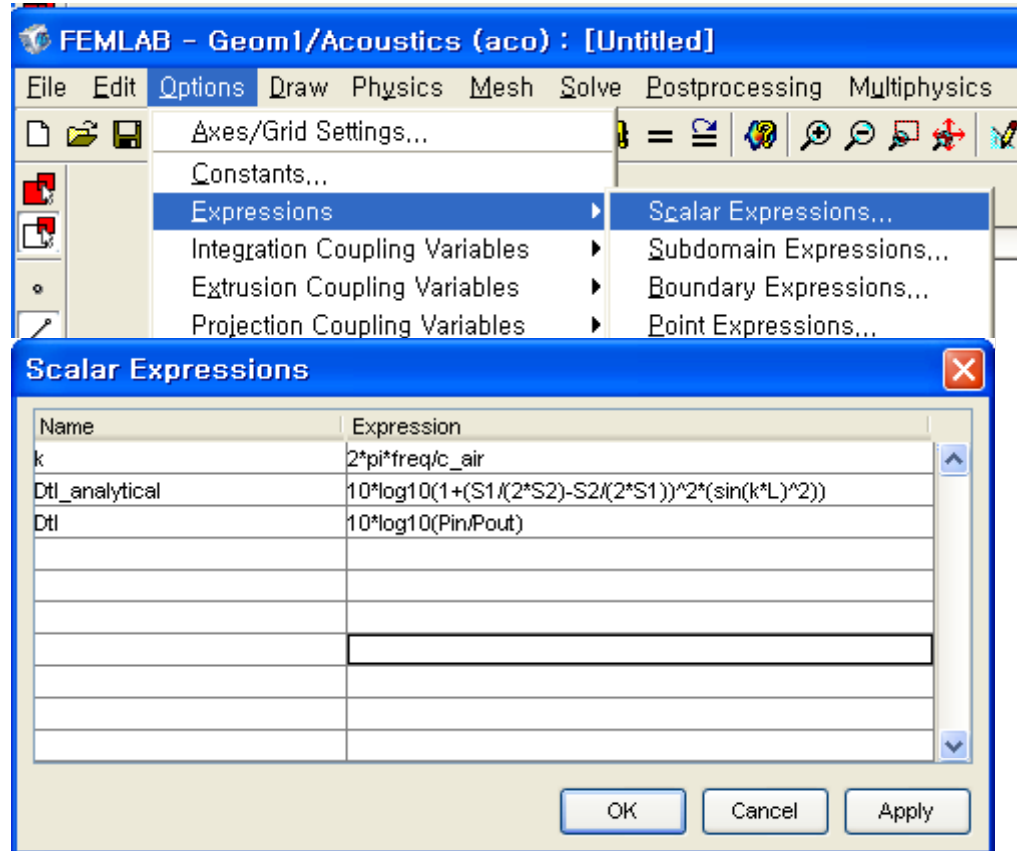
# Modeling using the FEMLAB

## Physics settings (boundary)



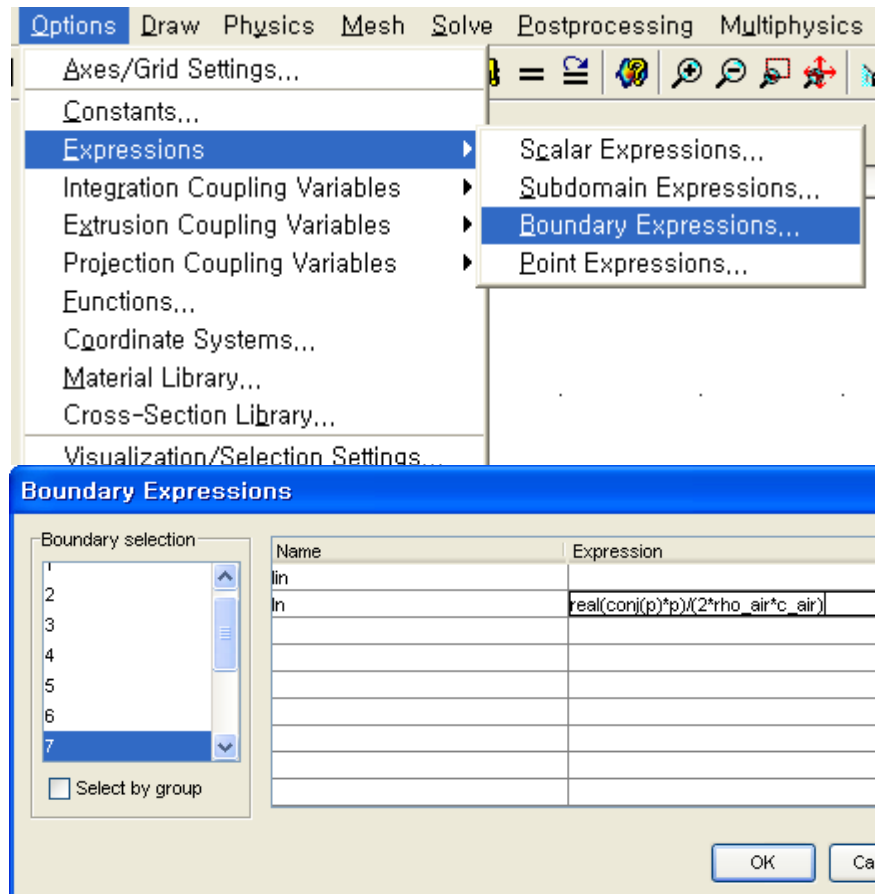
7. Select boundary 2 in the **Boundary selection** list.
8. Select **Radiation condition** in the **Boundary condition** list.
9. Type 1 in the **Pressure source** edit field.
10. Finally select boundary 7 in the **Boundary selection** list.
11. Select **Radiation condition** in the **Boundary condition** list.
12. Click **OK**.

# Modeling using the FEMLAB Physics settings (expression variables)



1. On the **Options** menu, point to **Expressions**, and then click **Scalar Expressions**.
2. In the **Scalar Expression** dialog box enter the following.
3. Click **OK**.

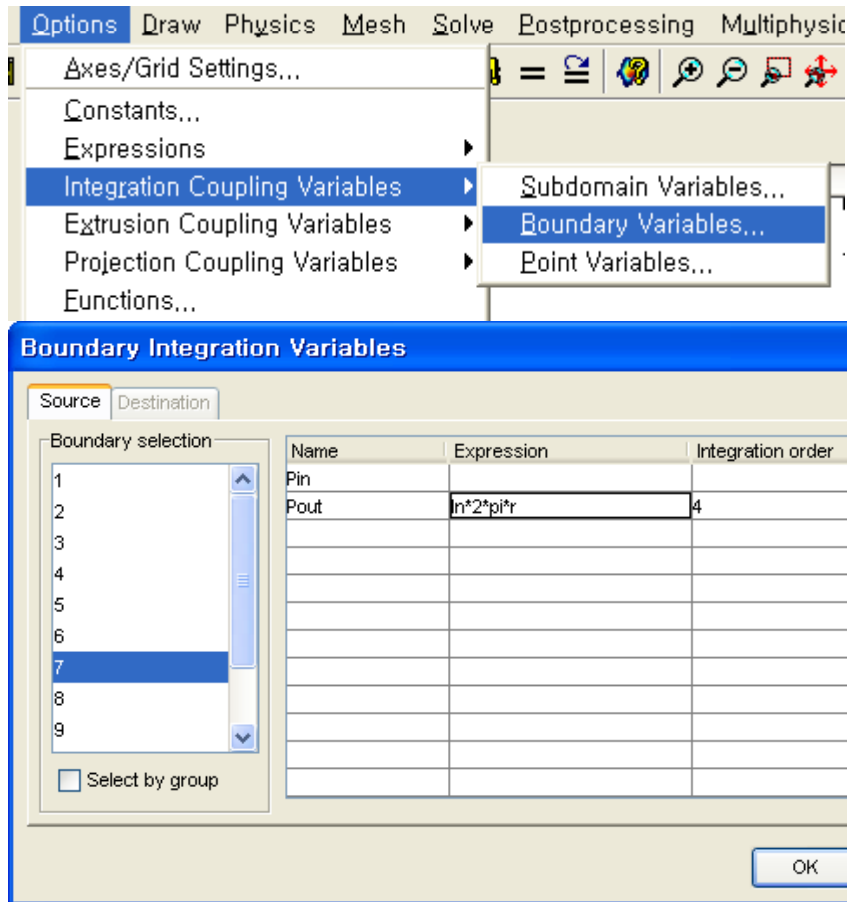
# Modeling using the FEMLAB Physics settings (expression variables)



4. Go the **Options** menu and choose **Expressions** and then **Boundary Expressions**.
5. In the **Boundary Expressions** dialog box select boundary 2 from the **Boundary selection** list and enter the following expression.
6. In the **Boundary Expression** dialog box select boundary 7 from the **Boundary selection list** and enter the following boundary integration variables
7. Click **OK**.

# Modeling using the FEMLAB

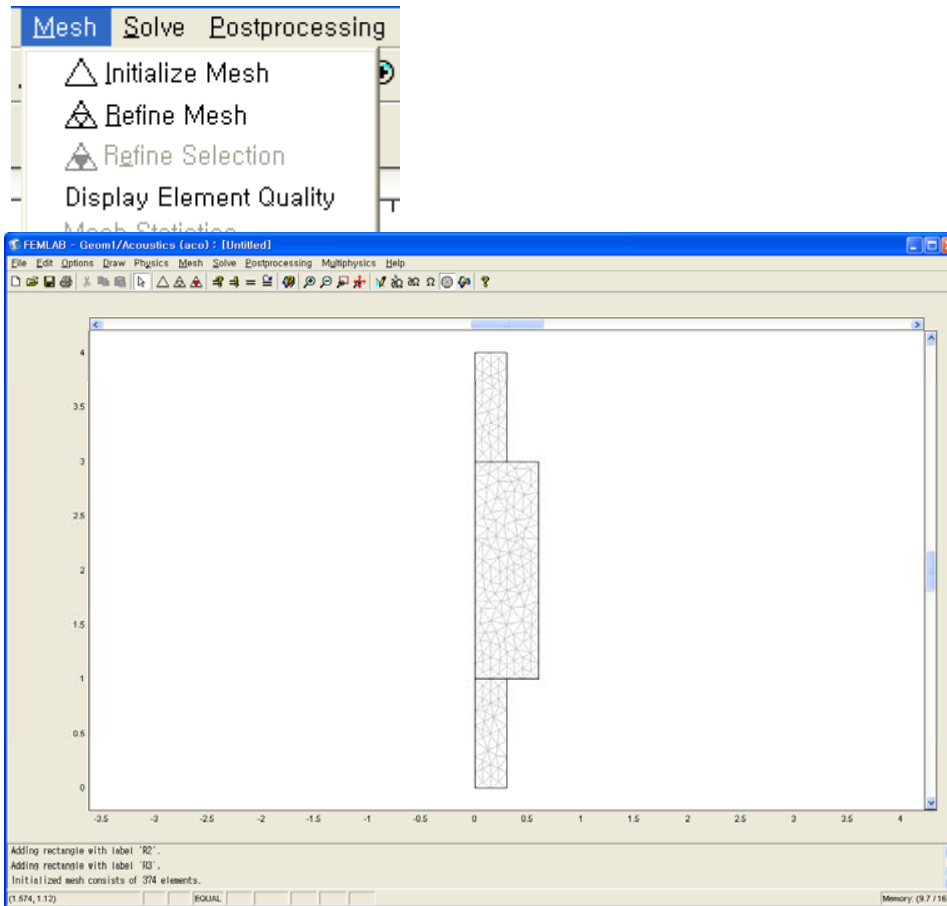
## Physics settings (scalar coupling variables)



1. Go to the **Options** menu and choose **Scalar Coupling Variables** and the **Boundary Variables**
2. In the **Boundary Integration Variables** dialog box select boundary 2 and then enter the following boundary integration expression.
3. In the **Boundary Integration Variables** dialog box select boundary 7 and enter the following boundary integration expression.
4. Click **OK**.

# Modeling using the FEMLAB

## Mesh generation

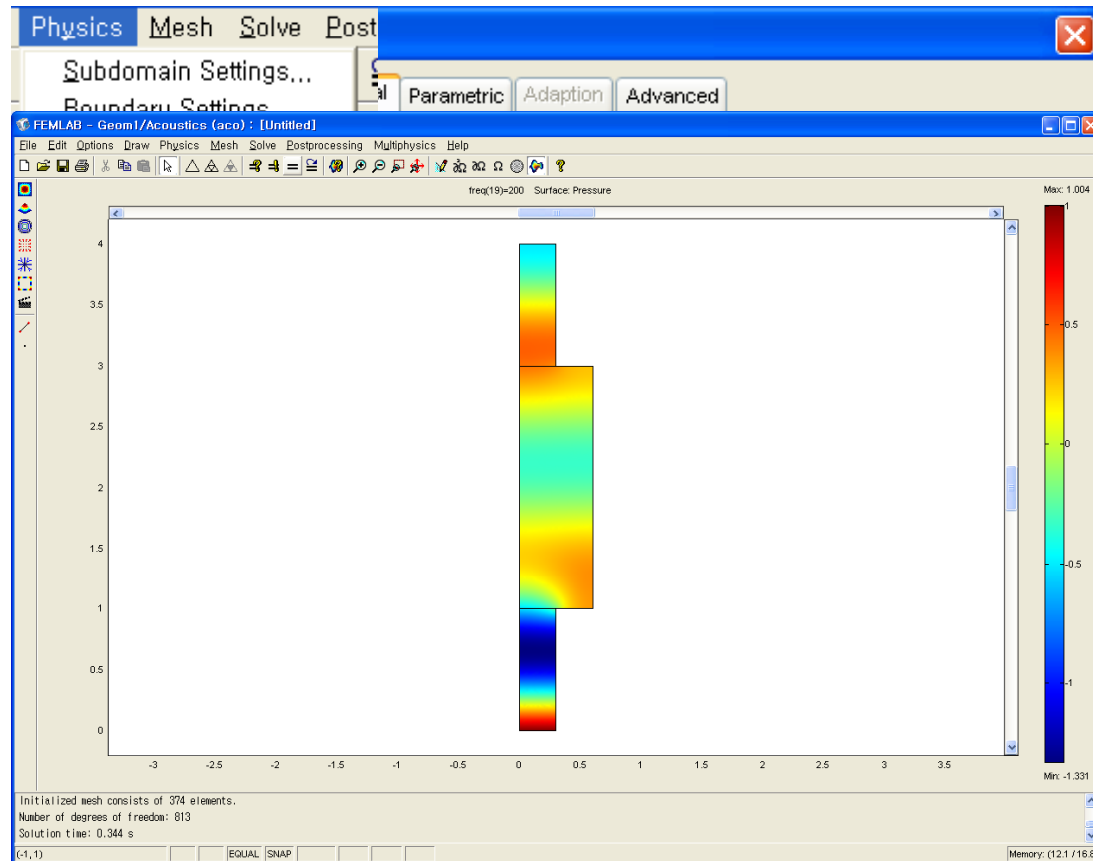


1. Go to the **Mesh** menu and choose **Mesh Parameters**.
2. In the **Mesh Parameters** dialog box select **Finer** from the **Predefined mesh size** list.
3. Click **Remesh**.
4. Click **OK**.



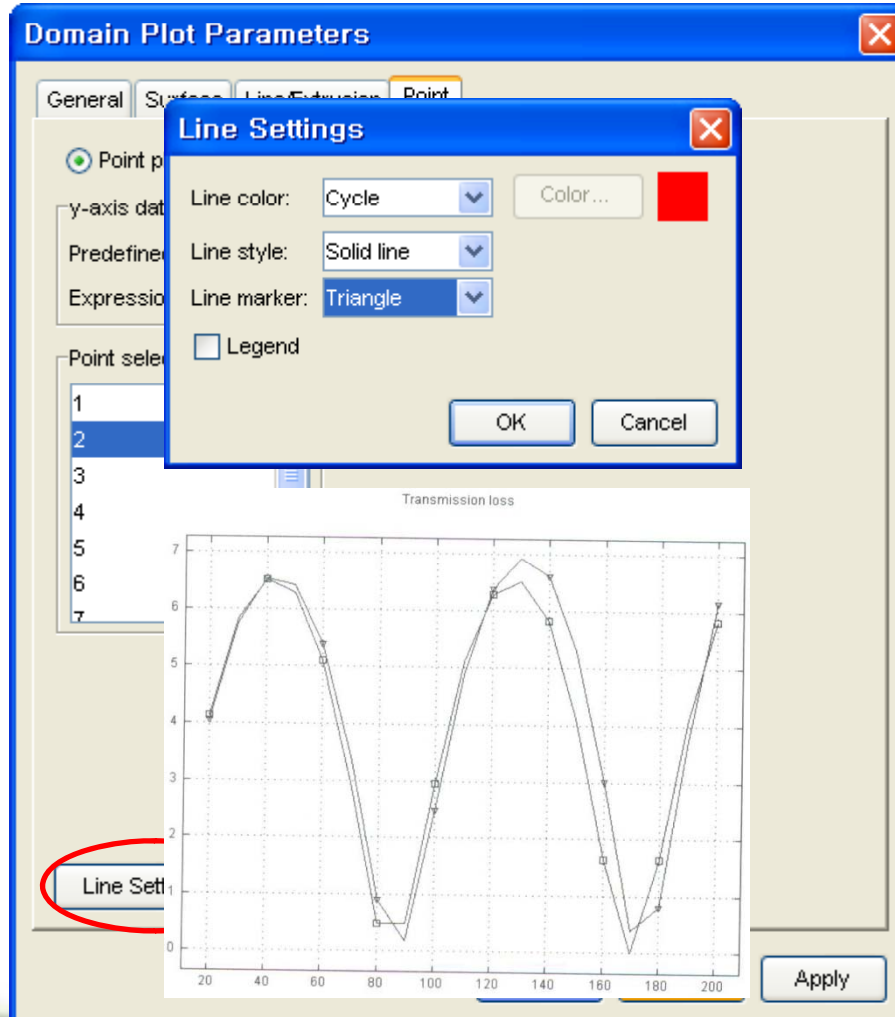
# Modeling using the FEMLAB

## Computing the solution



1. From the **Solve** menu, choose **Solver Parameters**.
2. In the **Solver Parameters** dialog box select **Parametric linear** from the solver selection list.
3. Enter **freq** in the **Name of Parameter** edit field.
4. Enter **20:10:200** in the **List of parameter values** edit field.
5. Click **OK**.
6. Go to the **Physics** menu and choose **Scalar Variables**.
7. Enter **freq** in the frequency edit field.
8. Click **OK**.
9. Click the **Solve** button to start the simulation.

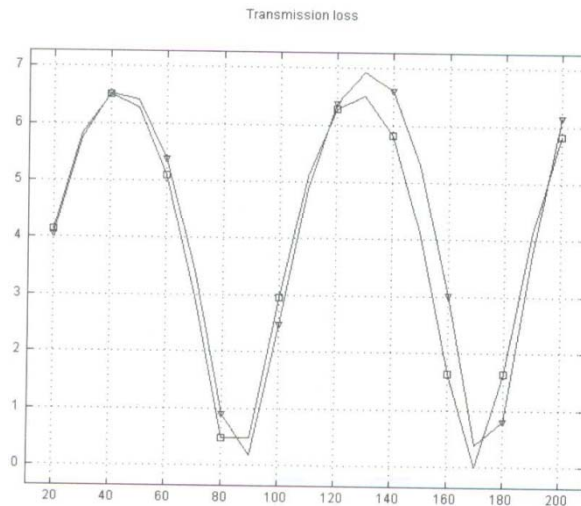
# Modeling using the FEMLAB Postprocessing and visualization



1. Go to the **Postprocessing** menu and select **Domain Plot Parameters**.
2. Click the **Keep current plot** page.
3. Click the **Title/Axis** button and enter **Transmission loss** in the **Title** edit field.
4. Click **OK**.
5. Select **Point plot** in **Domain Plot Parameters** dialog box and click on the **Point** tab.
6. Select point **1** from the **Point selection** list.
7. Enter **Dtl** in the **Expression** edit field.
8. Click the **Line Settings** button and select **Triangle** in the **Line marker** list.
9. Click **OK**.
10. Click **Apply** in the **Domain Plot Parameters** dialog box
11. **Dtl\_analytical** plot method is also same

# Conclusions

- The discrepancy increases with frequency between the 1D theoretical model and a 3D analysis.
- In the lower frequency range, there is good agreement between the theoretical solution and the FEM solution.



Dtl analysis : triangle

Dtl theoretical model : square