

Chapter 9. Agitation and Mixing of Liquids

Agitation (교반): the induced motion of a material in a circulatory pattern

Mixing (혼합): the random distribution of two or more separate phases

→ 원래 정의는 다르지만 많은 경우 혼용해서 사용

Purposes of agitation:

Suspending solid particles (suspension)

Blending miscible liquids (alcohol & water)

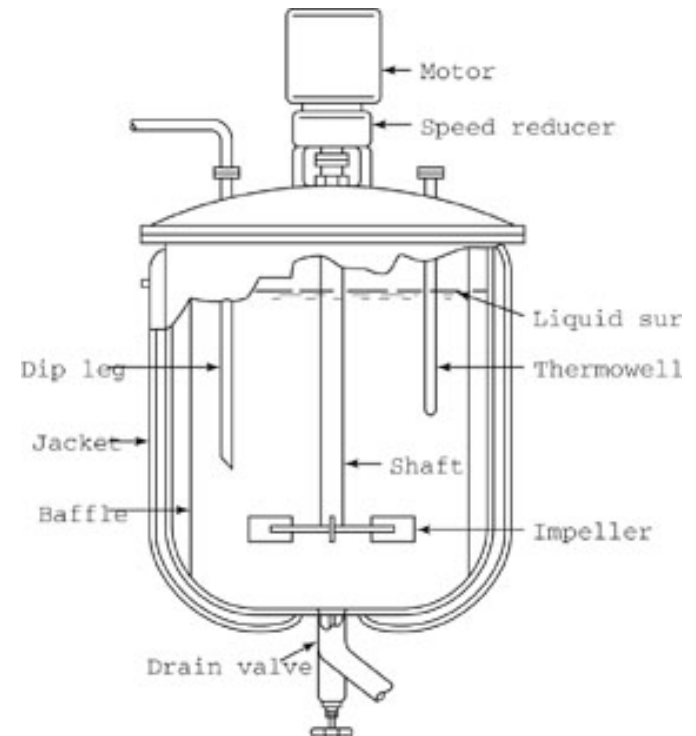
Dispersing a gas through the liquid (bubble)

Dispersing immiscible liquids (emulsion)

Promoting heat transfer

Agitated Vessels

: cylindrical form, vertical axis, closed or open top
round bottom, equal liquid depth & tank diameter

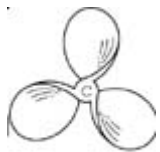
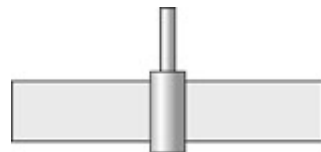
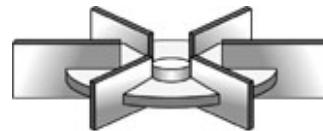


Typical agitation process vessel

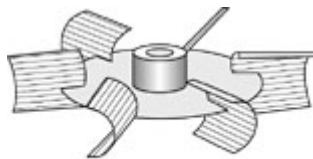
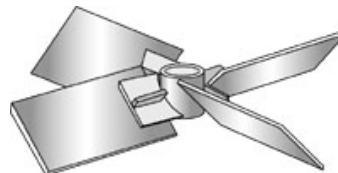
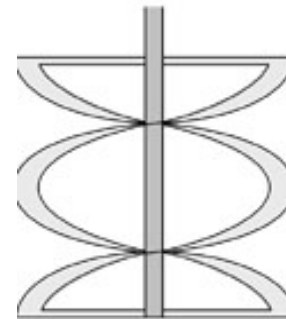
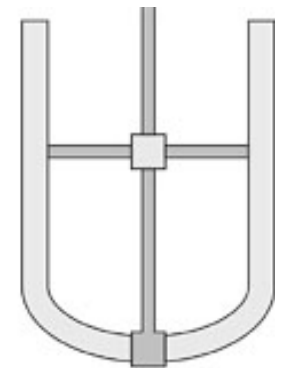
* Impellers

- { Axial-flow impellers (축류 임펠러)
- { Radial-flow impellers (방사류 임펠러)

- { Impellers for low- to moderate- viscosity liquids:
propellers, turbines & high efficiency impellers
- { Impellers for very viscous liquids:
helical impellers & anchor agitators

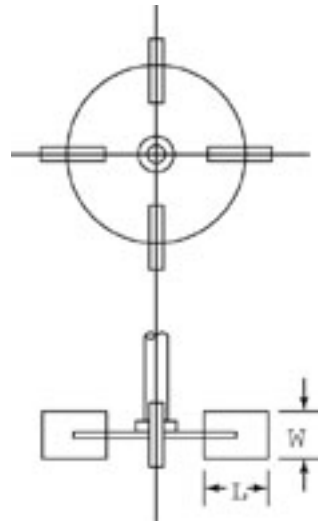
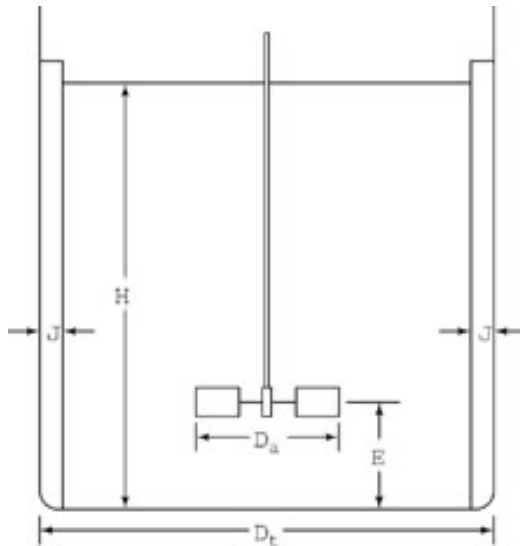
three-blade
propellerstraight-blade
turbine

disk turbine

concave-blade
disk turbinepitched-blade
turbinedouble-flight
helical-ribbon
impeller

anchor impeller

Standard turbine design

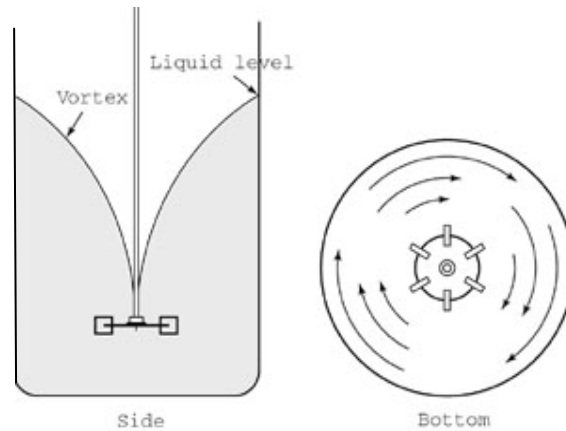
 H : depth of liquid D_t : tank diameter D_a : impeller diameter L : blade length W : impeller width J : width of baffle E : clearance

Typical proportions:

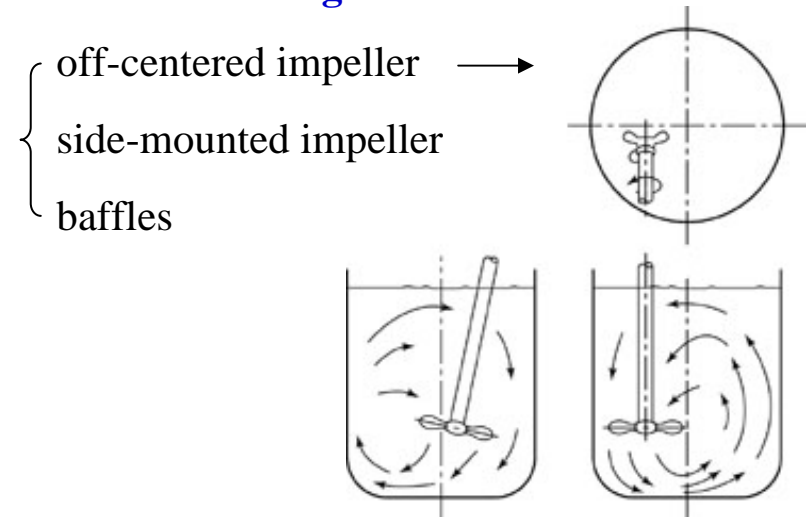
$$\frac{D_a}{D_t} = \frac{1}{3} \quad \frac{H}{D_t} = 1 \quad \frac{J}{D_t} = \frac{1}{12} \quad \frac{E}{D_t} = \frac{1}{3} \quad \frac{W}{D_a} = \frac{1}{5} \quad \frac{L}{D_a} = \frac{1}{4}$$

& No. of baffles: 4, No. of impeller blades: 6 or 8

Swirling flow pattern

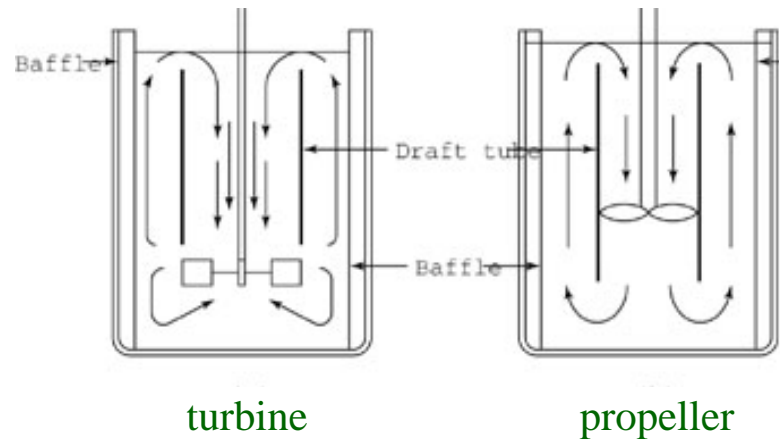


Prevention of swirling



Draft tubes

Controls direction and velocity of flow
Useful when high shear is desired such as emulsions and suspensions

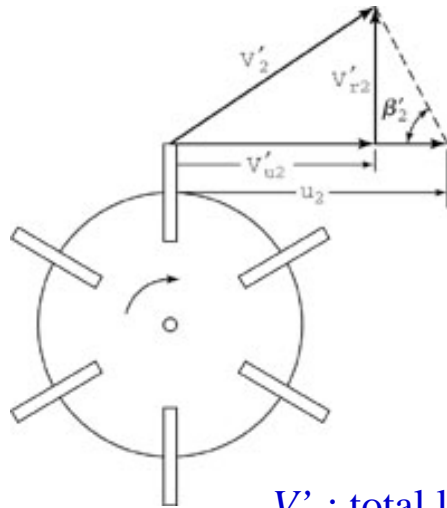


Draft tubes, baffled tank

* Circulation rates

Large impellers at medium speed → promotes flow

Smaller impellers at high speed → generates intense turbulence

Flow number, N_Q 

V'_2 : total liquid velocity

V'_{u2} : tangential velocity

V'_{r2} : radial velocity

u_2 : blade tip velocity

Volumetric flow rate through the impeller

$$q = V'_{r2} A_p$$

Eq. (9.5)
 $\propto \pi D_a n$

area of the cylinder
swept by impeller tip

$$= \pi D_a W$$

\downarrow
 $\propto D_a$

$$\Rightarrow \boxed{q \propto n D_a^3} \quad \text{--- Eq. (9.7)}$$

rotational speed

The rate of these two quantities → Flow number, N_Q

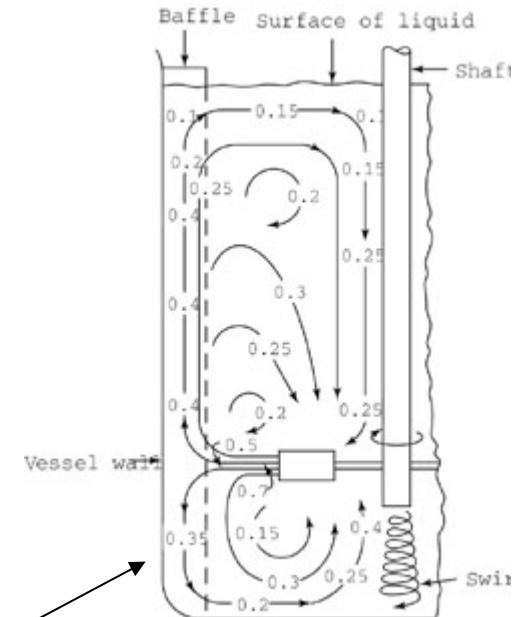
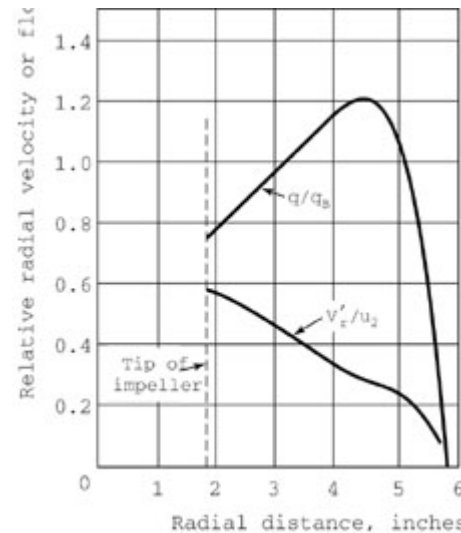
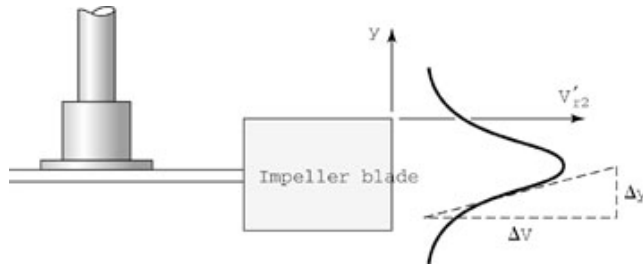
$$N_Q \equiv \frac{q}{n D_a^3} \quad \text{--- Eq. (9.8)}$$

For a disk turbine: $N_Q=1.3$

For marine propellers: $N_Q=0.5$

For a four-blade 45° turbine: $N_Q=0.87$

* Velocity patterns



Velocity profile and patterns in turbine agitator

Numbers indicate fractions of the velocity of the blade tip.

Tank Reynolds number, Re:

$$\boxed{\text{Re} = \frac{n D_a^2 \rho}{\mu}} \quad \longleftarrow \quad \text{Re} = \frac{D_a u_2 \rho}{\mu} = \frac{D_a (n D_a) \rho}{\mu}$$

\uparrow
 $u_2 \propto \pi D_a n$

u_2 : blade tip velocity

At $\text{Re} < 10$, laminar flow

At $\text{Re} > 10^4$, turbulent flow

*** Power consumption**

Power P : product of the flow rate q and the kinetic energy per unit volume E_k

$$q = n D_a^3 N_Q \quad E_k = \frac{\rho (V'_2)^2}{2}$$

$\longleftarrow V'_2 = \alpha u_2 = \alpha \pi n D_a$

$$P = n D_a^3 N_Q \frac{\rho (\alpha \pi n D_a)^2}{2}$$

$$= \rho n^3 D_a^5 \left(\frac{\alpha^2 \pi^2}{2} N_Q \right)$$

이를 무차원 형태(dimensionless form)로 고쳐 표현하면,

$$\frac{P}{n^3 D_a^5 \rho} = \frac{\alpha^2 \pi^2}{2} N_Q \quad \text{--- Eq. (9.11)}$$

윗 식에서 좌변 항을 Power number (동력수)로 정의함.

Power number, N_P :

$$N_P \equiv \frac{P}{n^3 D_a^5 \rho} \quad \text{--- Eq. (9.12a)}$$

: ratio of drag force to momentum flow

N_P is analogous to f or C_D

즉, N_P 가 크면 동력소비가 큼.

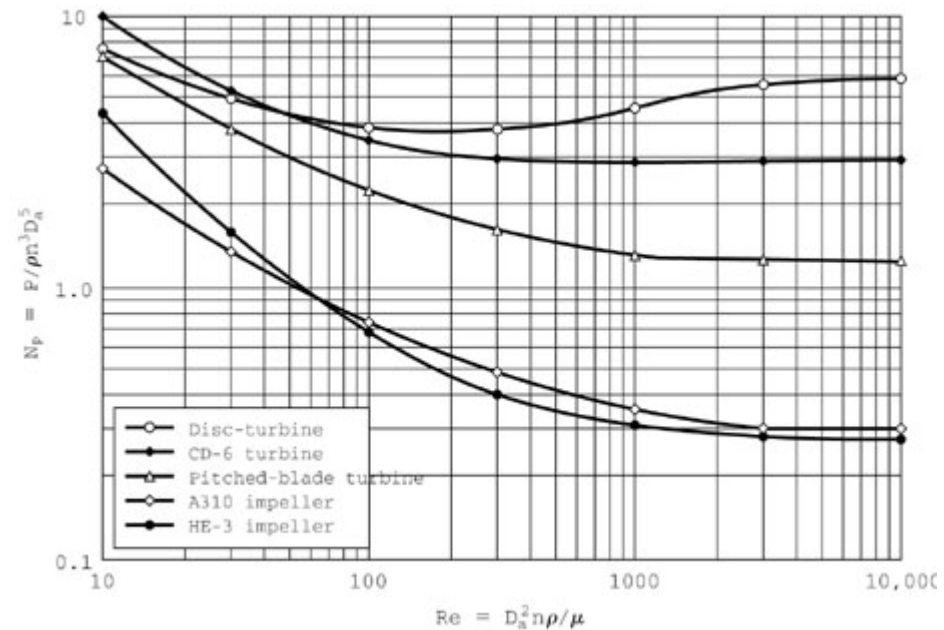


Fig. 9.13 Plots of power number N_P vs. Reynolds number Re for baffled tanks

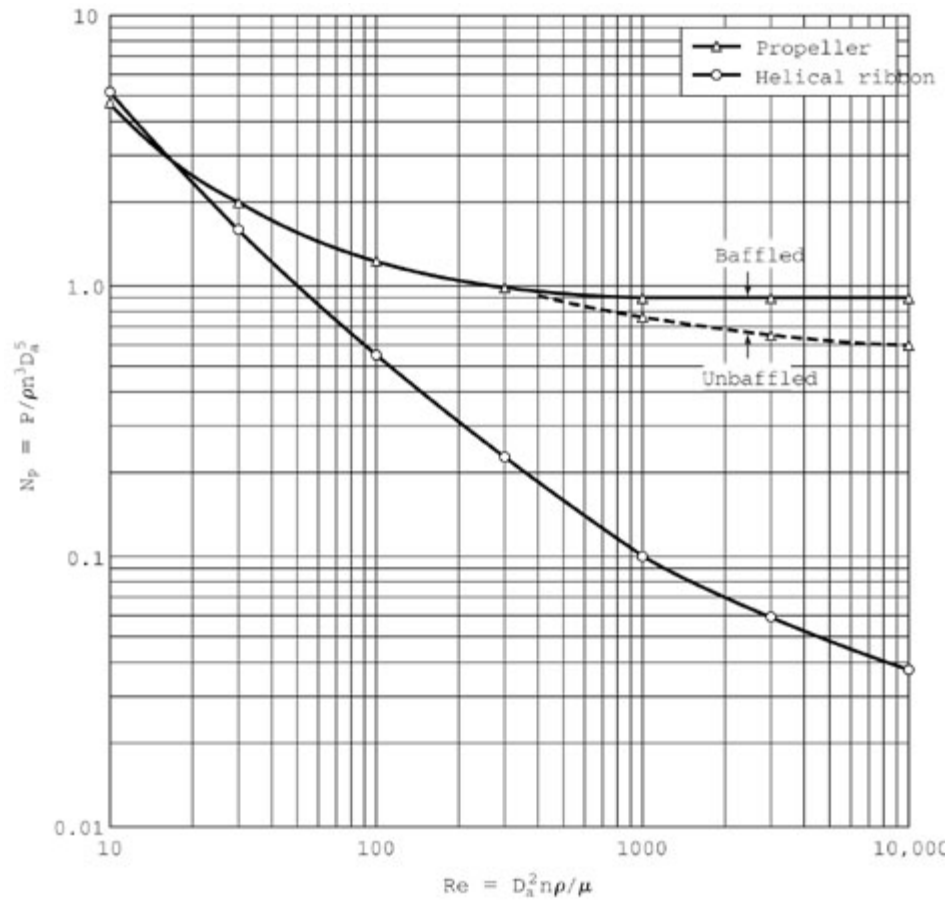


Fig. 9.14 Plots of power number N_p vs. Reynolds number Re for propellers and helical ribbons

Effect of system geometry

$$\cdot \frac{D_a}{D_t} \downarrow \longrightarrow \begin{cases} N_P \uparrow & \text{when baffles are few \& narrow} \\ N_P \downarrow & \text{when " many \& wide} \end{cases}$$

$$\cdot \frac{E}{D_t} \uparrow \longrightarrow \begin{cases} N_P \uparrow & \text{for a disk turbine} \\ N_P \downarrow & \text{for a pitched-blade turbine} \end{cases}$$

- Two turbines on the same shaft

$$\longrightarrow \begin{cases} 1.9 \text{ times power of one turbine when the spacing is longer than } D_a \\ 2.4 \text{ times " " " for closely spaced turbines} \end{cases}$$

- The shape of tank: little effect on N_P

Calculation of power consumption

The power delivered to the liquid,

$$P = N_P n^3 D_a^5 \rho \quad \text{--- Eq. (9.18)} \quad \leftarrow \text{ from Eq. (9.12a)}$$

At low Re ($Re < 10$),

$$N_P \equiv \frac{P}{n^3 D_a^5 \rho}$$

$$N_P = \frac{K_L}{Re} \quad \text{for both baffled \& unbaffled tanks}$$

$$\therefore P = K_L n^2 D_a^3 \mu \quad \text{--- Eq. (9.20)}$$

← ρ is not a factor

. 상수 K_L & K_T : Table 9.2
(또는 Figs. 9.13-14에서
 N_P 값으로 제공)

At high Re ($Re > 10,000$),

$$N_P \neq f(n, Re) \quad \text{for baffled tanks}$$

$$= K_T$$

$$\therefore P = K_T n^3 D_a^5 \rho \quad \text{--- Eq. (9.22)}$$

← μ is not a factor

Ex. 9.1) A disk turbine with 6 blades in a baffled tank 2 m in diameter

Turbine diameter of 0.67 m positioned 0.67 m above the tank bottom

Turbine blade width of 134 mm, A depth of 2 m with 50% NaOH solution

Viscosity of 12 cP, Density of 1,500 kg/m³, Impeller speed of 90 rpm

What power will be required?

Ans.)

$$Re = \frac{n D_a^2 \rho}{\mu} = \frac{1.5 (0.67)^2 1500}{0.012} \approx 84,169$$

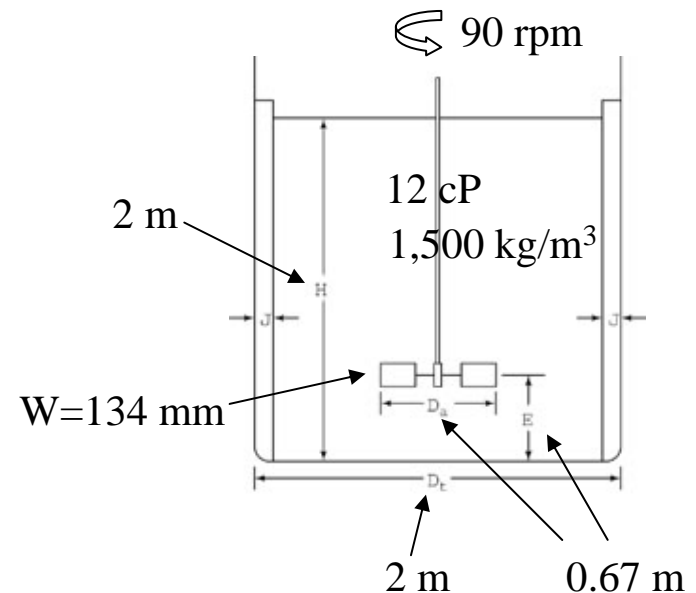
Re > 10,000 이므로 $N_p = K_T$ 사용

Table 9.2에서 $K_T = 5.75$

Eq. (9.22)에서 계산하면,

$$\begin{aligned} P &= K_T n^3 D_a^5 \rho = 5.75 \times 1.5^3 \times 0.67^5 \times 1,500 \\ &= \underline{\underline{3,930 \text{ W}}} \end{aligned}$$

μ is not a factor



Ex. 9.2) A disk turbine with 6 blades in a baffled tank 2 m in diameter

Turbine diameter of 0.67 m positioned 0.67 m above the tank bottom

Turbine blade width of 134 mm, A depth of 2 m with a rubber-latex compound

Viscosity of 120 Pa·s, Density of 1,120 kg/m³, Impeller speed of 90 rpm

What power will be required?

Ans.)

$$Re = \frac{nD_a^2 \rho}{\mu} = \frac{1.5(0.67)^2 1120}{120} \approx 6.3$$

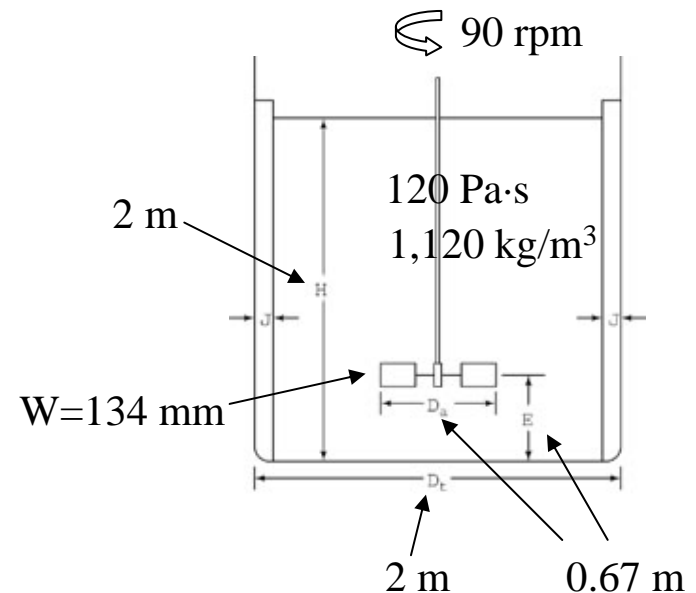
Re < 10 이므로 $N_p = K_L / Re$ 사용

Table 9.2에서 $K_L = 65$

Eq. (9.20)에서 계산하면,

$$\begin{aligned} P &= K_L n^2 D_a^3 \mu = 65 \times 1.5^2 \times 0.67^3 \times 120 \\ &= \underline{5,278 \text{ W}} \end{aligned}$$

ρ is not a factor



Blending and Mixing

Mixing time (혼합시간) t_T : the time to reach complete mixing (99% mixing)

← achieved if the contents of the tank are circulated about 5 times

$$t_T \approx \frac{5V}{q_T}$$



nt_T vs. Re

(Reynolds 수에 따른 혼합시간)

V : liquid volume in tank

q_T : total liquid flow rate

n : rotational speed (r/s)

ex) For turbine in a baffled tank

with $D_a/D_t=1/3$, $D_t/D_H=1$

→ $nt_T=36$ for $Re > 2,000$

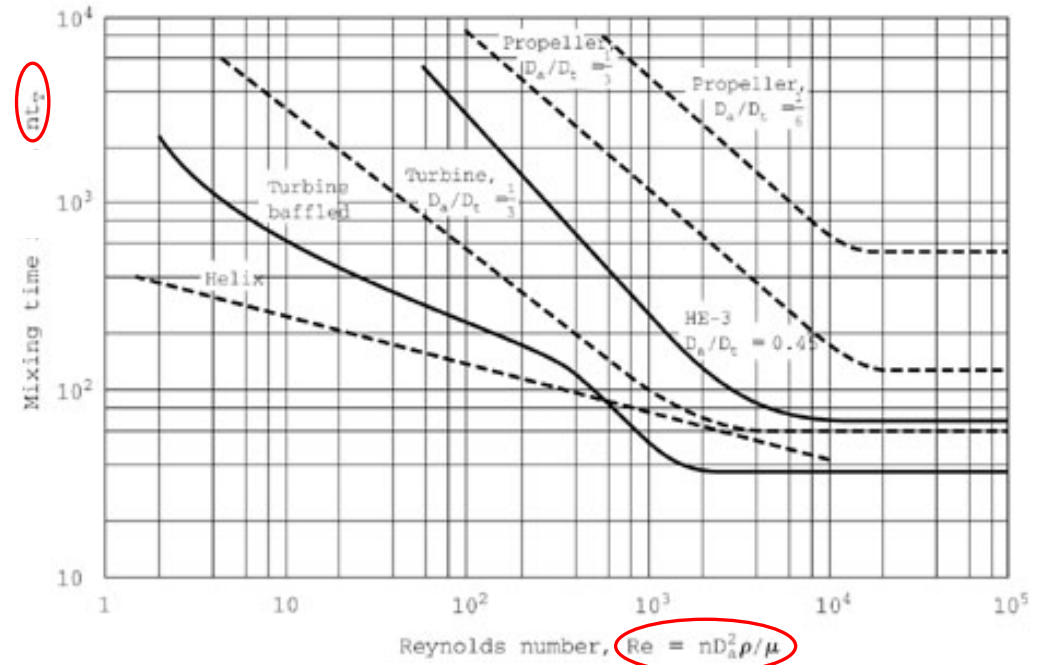


Fig. 9.16. Mixing times in agitated vessels.
Dashed lines for unbaffled tanks; Solid lines for baffled tanks.

Mixing time factor (혼합시간 인자) f_t

A general correlation for turbines:

$$f_t = nt_T \left(\frac{D_a}{D_t} \right)^2 \left(\frac{D_t}{H} \right)^{1/2} \left(\frac{g}{n^2 D_a} \right)^{1/6}$$

A helical ribbon agitator:

shorter mixing times with very viscous liquids

In a pseudoplastic liquid: blending time is much longer than in Newtonian liquids.

Ex. 9.3) $D_t = 6$ ft (1.83 m), 6-straight blade turbine, $D_a = 2$ ft (0.61 m), $E = D_a$, $n = 80$ rpm, $H = D_t$

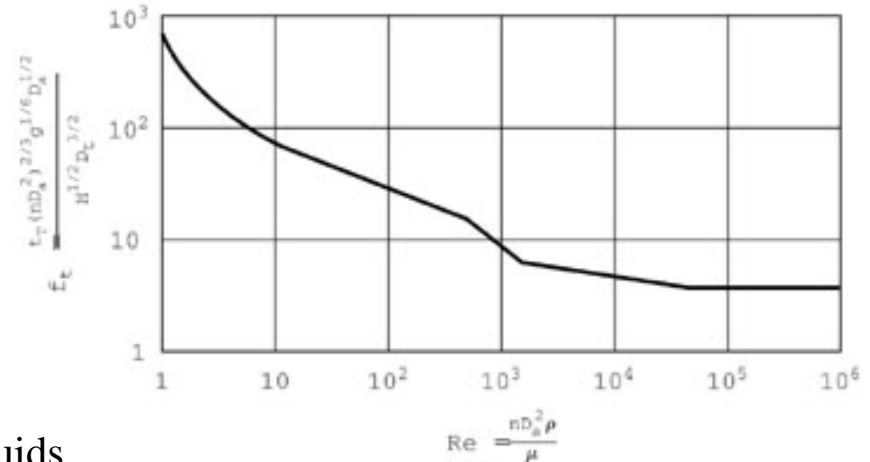
For neutralizing a NaOH solution with HNO₃ solution at 70 °F, $t_T = ?$

Ans.) From Appendix 6 $\rightarrow \rho = 62.30 \text{ lb/ft}^3 = 62.30(0.453 \text{ kg/lb})(\text{ft}/0.305 \text{ m})^3 = 997 \text{ kg/m}^3$

$$\mu = 0.982 \text{ cP} = 9.82 \times 10^{-4} \text{ Pa} \cdot \text{s}$$

$$\therefore \text{Re} = \frac{nD_a^2 \rho}{\mu} = 503,000$$

From Fig. 9.16 for $\text{Re} = 503,000$, $nt_T = 36 \rightarrow t_T = 36/1.333 = 27 \text{ s}$



Dispersion Operations

Volume (or holdup) of dispersed phase Ψ

$$\Psi = \frac{\pi N D_p^3}{6}$$

N : the number of drops or bubbles per total volume

Total surface area of drops per total volume a

$$a = \pi N D_p^2$$

$$\longrightarrow D_p = \frac{6\Psi}{a}$$

(실제로는 입자의 크기가 다르므로
평균입자경으로 정의)

Sauter mean diameter \bar{D}_s (or volume-surface mean diameter D_{32})

$$\bar{D}_s \equiv \frac{6\Psi}{a}$$

* Liquid/liquid dispersion

Weber number We

$$We = \frac{\rho_c (n D_a)^2}{\sigma / D_a} = \frac{\rho_c n^2 D_a^3}{\sigma}$$

density of continuous phase

: kinetic energy/surface energy

interfacial tension

$$\bar{D}_s / D_a \propto We^{-0.6} \longrightarrow We \uparrow \rightarrow \bar{D}_s \downarrow$$

Related problems: (Probs.) 9.1, 9.3, 9.5, 9.11(a) and 9.18